

**APPLICATION**  
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**TITLE:           POWER TRANSFER APPARATUS OF FOUR-  
WHEEL DRIVE VEHICLE AND DEVICE FOR  
CONTROLLING TORQUE**

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POWER TRANSFER APPARATUS OF FOUR-WHEEL DRIVE VEHICLE AND  
DEVICE FOR CONTROLLING TORQUE

BACKGROUND OF THE INVENTION

5           The present invention relates to an automobile power transfer apparatus for transferring to driving wheels power which is output from an engine, by way of a transmission. Further, the present invention also relates to a device for controlling torque to be  
10 transmitted to a clutch.

          A power transfer apparatus such as that shown in, e.g., Figs. 12 and 13, has hitherto been available as a power transfer apparatus of this type. Fig. 12 is a skeleton plan view of a four-wheel drive vehicle; and Fig.  
15 13 is a semi-cross-sectional view of a power transfer apparatus. As shown in Fig. 12, the power transfer apparatus 103 is interposed between drive shafts 105, 107 in the four-wheel drive vehicle 101.

          Driving force output from an engine 109 by way of a  
20 transmission and a transfer 111 is transmitted to rear wheels 117 by way of the drive shaft 105, the power transfer apparatus 103, the drive shaft 107, a rear differential 113, and an axle shaft 115. Meanwhile, the driving force is also transmitted to front wheels 123 by  
25 way of a front differential 119 and an axle shaft 121.

The power transfer apparatus 103 has such a structure as shown in Fig. 13. Specifically, a main clutch 129 serving as clutch means, a pilot clutch 131, and a cam mechanism 123 serving as conversion means are  
5 interposed between an inner rotary member 125 and an outer rotary member 127. The power transfer apparatus 103 has an electromagnet 135. The outer rotary member 127 is joined to the drive shaft 105 (see Fig. 12), and the inner rotary member 125 is joined to the other drive  
10 shaft 107 (see Fig. 12).

When the pilot clutch 131 is engaged by means of electromagnetic force derived from energization control of the electromagnet 135, the cam mechanism 123 is actuated, thereby rendering the main clutch 129 engaged.  
15 Consequently, the driving force transmitted to the drive shaft 105 is transmitted to the outer rotary member 127, and torque is transferred to the internal rotary member 125 by way of the main clutch 129, whereupon the driving force is transferred to the other drive shaft 107.

20 In this way, the four-wheel drive vehicle 101 shown in Fig. 12 enables transmission, to the rear wheels 117, of the driving force output from the engine 109 by way of the transmission and the transfer 111, by means of fastening force control embodied by energization control  
25 of the power transfer apparatus 103 disposed on and

between the drive shafts 105, 107 (refer to, e.g., Japanese Patent Unexamined Publication No. 2000-234633, particularly, pages 2 to 4 and Figs. 1 and 3 thereof.)

The energization control of the electromagnet 135 of the power transfer apparatus 103 is performed in accordance with an accelerator position or a throttle position of the automobile. Figs. 14 and 15 are timing charts showing variations in target values of torque to be transmitted by the power transfer apparatus 103 (hereinafter referred to as "transmission torque") in terms of an accelerator position or a throttle position and an A/T (automatic transmission) range. Figs. 14 and 15 show conditions of the vehicle when switching is made from a forward traveling state to a rearward traveling state. Fig. 15 shows a state in which the four-wheel drive vehicle remains stationary by means of so-called tight corner braking phenomenon. Figs. 14A and 15A show variations in accelerator position or throttle position, Figs. 14B and 15B show a change in the A/T range; and Figs. 14C and 15C show variations in target torque (control current value).

First, the four-wheel drive vehicle shown in Fig. 12 is taking corners under the stipulation that a tight corner braking phenomenon will arise. A friction factor  $\mu$  existing between tires and the road surface is high.

Further, the gear position of the transmission; that is, an A/T range of an automatic transmission in this example, is, for example, a drive range D. The accelerator or throttle position is in an arbitrary open position.

5           In such a traveling state, when the A/T range is switched from the drive range D to a reverse range R at a point in time (3) shown in Fig. 14, the accelerator or throttle position is actuated from an arbitrary open position to an arbitrary closed position at a point in  
10 time (1) (tip-out) and actuated from an arbitrary close position to an arbitrary open position at a point in time (2) (tip-in). Switching between the tip-out and the tip-in is effected within an extremely short period of time.

          After the tip-out (1) has arisen, the target torque  
15 (i.e., a target value of transmission torque) of the power transfer apparatus 103 is transmitted along a line segment F1 at, e.g., a given slope. For instance, energization of the electromagnet 135 is controlled such that the target torque decreases to, e.g., transmission  
20 torque of 0 Nm.

          However, when energization control is performed along such a line segment F1, the transmission torque of the power transfer apparatus 103 fails to reach a predetermined level during a duration from the tip-out  
25 (1) to the tip-in (2), the duration being an extremely

short period of time. The control current again increases from a point A so as to attain the target torque corresponding to an arbitrary accelerator or throttle position.

5           When the gear-shifting direction of the transmission is inverted during the period existing between the tip-out (1) and the tip-in (2); for example, when switching is made from a drive position (driving forward) to a reverse position (driving backwards), with the  
10 transmission torque still remaining in the power transfer apparatus 103 in the manner as mentioned previously, there arises a problem of occurrence of an unusual noise in the power transfer apparatus 103, the transfer 111, the rear differential 113, or the front differential 119.

15           Specifically, when the gearing direction of the automatic transmission is inverted as a result of the automatic transmission having been switched from the drive position to the reverse position with the transmission torque still being ensured in the power  
20 transfer apparatus 103, the direction of torsion of the drive shafts 105, 107 of the drive system existing between the front and rear wheels 123, 117 is changed by means of the tight corner braking phenomenon under the foregoing driving conditions. If a change arises in the  
25 direction—in which backlash is eliminated along the axis

of the power transfer apparatus 103—or the torque  
(force)—which would arise at the time of elimination of  
backlash—exceeds a predetermined level because of a  
change in the direction of torsion, there will arise a  
5 problem of occurrence of a collision sound grating on the  
occupants' ears or physical shock that the occupants find  
offensive.

Such collision sound or physical shock is induced by  
means of controlling the target torque of the power  
10 transfer apparatus 103 along the line segment F1 having  
a gentle slope. If the target torque is controlled along,  
e.g., a line segment F2 having a steep slope, the power  
transfer apparatus 103 can achieve a predetermined value  
of transmission torque before the tip-out (2). Therefore,  
15 in this case, even if a change has arisen in the  
direction of torsion in the drive shafts 105, 107,  
unusual noise which would grate on the occupants' ears  
will not arise as compared with the case where the  
transmission torque still remains in the power transfer  
20 apparatus 103.

In contrast, if the target torque of the power  
transfer apparatus 103 is controlled along the line  
segment F2 having a steep slope, vibration will arise in  
the vicinity of a range in which the transmission torque  
25 of the power transfer apparatus 103 assumes a value of 0

Nm. Hence, there arises a problem of the vibration being transmitted to occupants as physical shock, such as swaying motion.

According to the traveling condition illustrated in Fig. 15, the vehicle is taking corners under the stipulation that a tight corner braking phenomenon will arise, with the accelerator or throttle position being set in an arbitrary open position. The friction factor  $\mu$  existing between tires and the road surface is high. Further, there will arise a situation in which the vehicle comes to a halt while taking corners by means of the tight corner braking phenomenon. In such a situation where the vehicle remains stationary, the A/T range of the automatic transmission can be considered to be switched from the drive range D to the reverse range R; i.e., inversion of the gear-shifting direction can arise. At this time, the target torque of the power transfer apparatus 103 is presumed to be set to an accelerator or throttle position required to make such a setting that the target torque exceeds an allowable level at which no unusual noise arises.

Accordingly, inversion of the gear-shifting direction arises in the same situation in which the transmission torque exists in the power transfer apparatus 103, in the same manner as described in



connection with Fig. 14. The direction of torsion of the drive shafts 105, 107 is changed by the tight corner braking phenomenon in the same manner as mentioned above. Similarly, the torque developing at the time of inversion  
5 of the direction in which backlash is eliminated exceeds the allowable level at which no unusual noise arises, and hence there will arise a problem of occurrence of unusual noise (a collision sound) grating on the occupants' ears that the occupants find offensive.

10       The descriptions have described inversion of the gear-shifting direction by reference to the operation for switching the drive range D to the reverse range R. The inversion of gear-shifting direction arises in the same manner even by means of an operation for switching the  
15 reverse range R to the drive range D, thereby involving occurrence of unusual noise in the same manner.

      In relation to the traveling state of the vehicle (i.e., a transmission state of driving force to the vehicle) and the gear-shifting state of the transmission,  
20 such as a request for high torque at a low speed of the vehicle or a request for low torque at a high speed of the vehicle, driving stability must be improved.

#### SUMMARY OF THE PRESENT INVENTION

25       It is a first object of the present invention to

inhibit occurrence of unusual sound by paying attention to unusual sound which arises in accordance with inversion of the gear-shifting direction of a transmission. In addition to this, it is a second object of the present invention to inhibit occurrence of unusual sound by reducing physical shock which would arise at the time of a tip-out. Further, it is also an object of the present invention to appropriately transmit to a vehicle transmission driving force, which is produced in a driving state including a halt and start of the vehicle, thereby improving driving stability.

The above-mentioned object can be achieved by a power transfer apparatus of a four-wheel drive vehicle according to a first aspect of the present invention.

15 The power transfer apparatus is disposed between front and rear wheels and 1) is capable of transmitting, to the wheels, driving force output from an engine through a transmission in accordance with an engagement force of a clutch member while the engagement force of clutch member is controlled, or 2) is capable of transmitting, to the wheels, driving force output from an engine through a transmission, while a differential motion of a differential disposed between the front and rear wheels is limited by controlling an engagement force of a clutch members. (In other words, the power transfer apparatus

25

is disposed between primary and secondary wheels, and has  
1) a clutch member being capable of distributing a torque  
(i.e, a driving force), that is generated by an engine  
via a transmission, between the primary wheel to the  
5 secondary wheel directly, or 2) a clutch member and a  
differential being capable of distributing a torque, that  
is generated by an engine via a transmission, between the  
primary wheel to the secondary wheel while a differential  
motion of the differential disposed between the primary  
10 wheel and the secondary wheel is limited by controlling  
an engagement force of the clutch member.)

The transmission comprises:

inversion detection member for detecting inversion  
or non-inversion of a gear-shifting direction of the  
15 transmission; and

control member for effecting control operation such  
that torque to be transmitted by the clutch members is  
decreased to a target value in accordance with an  
inversion signal pertaining to the gear-shifting  
20 direction detected by the inversion detection member.

In the above-mentioned power transfer apparatus of  
a four-wheel drive vehicle, according to a second aspect  
of the present invention, it is preferable that the  
control member performs control operation by setting a  
25 target control value corresponding to the target value of

the torque to be transmitted, thereby maintaining the target control value for a given period of time.

In the above-mentioned power transfer apparatus of a four-wheel drive vehicle, according to a third aspect of the present invention, it is more preferable to, further comprise: accelerator or throttle position detection member for detecting whether the accelerator or throttle position is in an arbitrary open position or an arbitrary closed position, wherein, when an inversion signal detected by the inversion detection member arises during the course of the accelerator or throttle position detected by the accelerator or throttle position detection member shifting again to an open position after having shifted from an arbitrary open position to an arbitrary closed position, the control member sets the given period of time on the basis of the inversion signal before the accelerator or throttle position again shifts to the open position.

In the above-mentioned power transfer apparatus of a four-wheel drive vehicle, according to a fourth aspect of the present invention, it is advantageous that, when an inversion signal detected by the inversion detection member arises during the course of the accelerator or throttle position detected by the accelerator or throttle position detection member shifting again to an open

position after having shifted from an arbitrary open position to an arbitrary closed position and when shifting of the accelerator or throttle position from an arbitrary open position to an arbitrary closed position  
5 has been detected, the control member performs control operation for reducing the torque to be transmitted by the clutch member and making a decrease rate steep at the beginning and then gentle before the inversion signal detected by the inversion detection member arises.

10 In the above-mentioned power transfer apparatus of a four-wheel drive vehicle, according to a fifth aspect of the present invention, it is further advantageous that the control member makes equivalent to a time constant the decrease rate of the torque to be transmitted.

15 Moreover, in the above-mentioned power transfer apparatus of a four-wheel drive vehicle, according to a sixth aspect of the present invention, it is preferable that the clutch member comprises a main clutch which is interposed between inner and outer rotary members, which  
20 enables transmission of torque by generating frictional engagement force when the inner and outer rotary members rotate in relation to each other, and which increases or decreases the frictional engagement force in accordance with axial pressing force; a pilot clutch which performs  
25 frictional engagement by member of electromagnetic force

generated through the energization control; and  
conversion member which operates by member of frictional  
engagement of the pilot clutch, which converts force  
derived from frictional engagement of the pilot clutch,  
5 and which imparts pressing force to the main clutch.

Further, the above-mentioned object can also be  
attained by a power transfer apparatus of a four-wheel  
drive vehicle according to a second aspect of the present  
invention. The power transfer apparatus is disposed  
10 between front and rear wheels and 1) is capable of  
transmitting, to the wheels, driving force output from an  
engine through a transmission in accordance with an  
engagement force of a clutch member while the engagement  
force of a clutch member may be controlled by the  
15 electromagnetic force derived from energization control,  
or 2) is capable of transmitting, to the wheels, driving  
force output from an engine through a transmission, while  
a differential motion of a differential disposed between  
the front and rear wheels is limited by controlling an  
20 engagement force of a clutch member. (In other words,  
the power transfer apparatus is disposed between primary  
and secondary wheels and has 1) a clutch member being  
capable of distributing a torque (i.e, a driving force),  
that is generated by an engine via a transmission,  
25 between the primary wheel to the secondary wheel directly,

or 2) a clutch member and a differential being capable of distributing a torque, that is generated by an engine via a transmission, between the primary wheel to the secondary wheel while a differential motion of the differential disposed between the primary wheel and the secondary wheel is limited by controlling an engagement force of the clutch member.)

The power transfer apparatus comprises: gear shift status detection member for detecting the state of gear shift of the transmission; and control member for subjecting an electromagnet to energization control such that the torque to be transmitted by the clutch member attains a target value in accordance with a gear shift status signal detected by the gear shift status detection member.

In the above-mentioned power transfer apparatus, it is advantageous that the torque is transmitted in accordance with an engagement force of the clutch while an engagement force of the clutch is controlled by an electromagnetic force derived from energization control.

Further, in the above-mentioned power transfer apparatus, it is also advantageous that the torque is transmitted while a differential motion of a differential disposed between the primary wheel and the secondary wheel is limited by controlling an engagement force of

the clutch that is controlled by the electromagnetic force derived from energization control.

Note that for example, the energization control can be conducted by controlling a current level or an  
5 amplitude of the clutch.

Further, note that although the clutch is controlled by the electromagnetic force derived from energization control in the above-mentioned aspects, it is possible to modify it in such a manner that the clutch can be  
10 controlled hydraulically.

According to the first aspect of the invention, the engagement force of the clutch member disposed between the front and rear wheels is controlled, and the driving force output from the engine by way of the transmission  
15 can be transmitted to the wheels while a limitation is imposed on the difference in movement between the front and rear wheels according to or by member of the engagement force. Further, the inversion detection member detects occurrence of inversion or non-inversion  
20 of gear-shifting direction of the transmission. In accordance with a signal pertaining to inversion of the gear-shifting direction detected by the inversion detection member, the control member can perform control operation so as to decrease to a target value the torque  
25 to be transmitted by the clutch member (transmission



torque).

Therefore, given that the clutch member possesses engagement force while the accelerator or throttle position is held in an arbitrary position and that the  
5 four-wheel drive vehicle is taking corners, if the gear-shifting direction of the transmission is switched from a forward driving direction to a rearward driving direction or from a rearward driving direction to a forward driving direction while the accelerator or throttle position is  
10 being shifted from an arbitrary open position to an arbitrary closed position and again to the open position or while the four-wheel drive vehicle remains stationary because of so-called tight corner braking phenomenon, the torque transmitted by the clutch member is decreased to a  
15 target value, e.g., 0 Nm, by member of the inversion signal. As a result, the transmission torque of the clutch member of the power transfer apparatus no longer remains or becomes nominal. Therefore, even when the direction of distortion of the drive system is changed by  
20 member of the tight corner braking phenomenon, to thereby change the direction in which backlash is to be eliminated along the axis of the power transfer apparatus, torque (force) resulting from elimination of backlash falls within an allowable value in which no  
25 unusual noise arises. Hence, occurrence of collision

sound, which would grate on the occupants' ears, is avoided, and occurrence of unusual noise can be inhibited.

In addition to yielding the effect yielded by the first aspect of the invention, the second aspect of the invention enables the control member to perform control operation by member of setting a target control value corresponding to the target value of the transmission torque, to thus maintain the target control value for a given period of time. Accordingly, when control is performed in accordance with the target control value, the transmission torque of the clutch member can be reduced to the target value without fail even if a lag has arisen in the response of the clutch member. Hence, occurrence of unusual noise can be inhibited reliably.

In addition to yielding the effect yielded by the second aspect of the present invention, the third aspect of the invention enables the accelerator or throttle position detection member to detect whether the accelerator or throttle is in an arbitrary open position or an arbitrary closed position. When an inversion signal to be detected by the inversion detection member arises during the course of the accelerator or throttle position to be detected by the accelerator or throttle position detection member being shift from an arbitrary open position to an arbitrary closed position and again

to the open position, the control member can set the given period of time on the basis of the inversion signal before the accelerator or throttle position shifts again to the open position.

5           Therefore, when the clutch member of the power transfer apparatus has achieved the target transmission torque with the accelerator or throttle position being in an arbitrary open position and when the vehicle is in a four-wheel drive state and taking corners, the  
10 transmission is switched from a drive range to a reverse range during a period between a tip-out and a tip-in of the accelerator or throttle position. When the gear-shifting direction is reversed, the transmission torque of the clutch member can be reduced to the target value  
15 without fail during a period starting from occurrence of the inverse signal and ending in occurrence of the tip-out. As a result, collision sound or physical shock, which grates on or becomes offensive to the vehicle's occupants and would otherwise arise at the time of  
20 occurrence of a tip-out, is prevented, thereby inhibiting generation of unusual noise or occurrence of physical shock.

          In addition to yielding the effect yielded by the the third aspect of the invention, the fourth aspect of  
25 the invention enables the control member to perform

control operation such that the transmission torque of the clutch member is reduced before the inverse signal to be detected by the inversion detection member arises and such that a decrease rate is made steep at the beginning  
5 and then gentle when an inversion signal detected by the inversion detection member arises during the course of the accelerator or throttle position detected by the accelerator or throttle position detection member shifting again to an open position after having shifted  
10 from an arbitrary open position to an arbitrary closed position and when the accelerator or throttle position has shifted from an arbitrary open position to an arbitrary closed position.

The control operation to be performed from when the  
15 tip-out has arisen until when the inverse signal has arisen can be made different from that to be performed from when the inverse signal has arisen until when the tip-in has arisen. Therefore, during a period from when the tip-out has arisen until when the inverse signal has  
20 arisen, there can be inhibited occurrence of physical shock, which would otherwise cause the vehicle body to vibrate because of a sharp drop in the control value to the target torque. Concurrently, the time required to diminish the transmission torque can be shortened.  
25 During a period from when the inverse signal has arisen

until when the tip-in has arisen, the control value is rapidly decreased so as to assume the target torque. The transmission torque of the clutch member is set to the target value without fail at the time of tip-out. Even  
5 when a change has arisen in the direction in which backlash is to be eliminated along the axis of the power transfer apparatus , for reasons of a change in the direction of torsion of the drive system due to the tight corner braking phenomenon, the torque (force) derived  
10 from elimination of backlash falls within an allowable value in which no unusual noise arises. Hence, there is inhibited occurrence of collision sound or vibration, which would otherwise grate on or become offensive to the occupants, thereby reliably preventing occurrence of  
15 unusual noise or physical shock.

In addition to yielding the effect yielded by the invention of the fourth aspect of the present invention, the fifth aspect of the invention enables the control member to make equivalent to a time constant the decrease  
20 rate of the torque to be transmitted. Therefore, an amplitude can be reduced in terms of vibrational response. Physical shock such as fluctuations in the vehicle body can be dampened more reliably.

In addition to yielding the effect yielded by the  
25 invention of any one of The first to fifth aspect of the

present invention, the sixth aspect of the invention enables the clutch member to comprise: a main clutch which is interposed between inner and outer rotary members, which enables transmission of torque by  
5 generating frictional engagement force when the inner and outer rotary members rotate in relation to each other, and which increases or decreases the frictional engagement force in accordance with axial pressing force; a pilot clutch which performs frictional engagement by  
10 member of electromagnetic force generated through the energization control; and conversion member which operates by member of frictional engagement of the pilot clutch, which converts force derived from frictional engagement of the pilot clutch, and which imparts  
15 pressing force to the main clutch. When the pilot clutch is subjected to frictional engagement by member of the electromagnetic force due to energization control according to the accelerator or throttle position, the conversion member converts the frictional engagement  
20 force, to thereby impart pressing force to the main clutch. In accordance with the pressing force, the main clutch can increase or decrease the frictional engagement force. By member of the main clutch, the frictional engagement force is produced when the inner and outer  
25 rotary members rotate in relation to each other, thereby

enabling transmission of torque between the inner and outer rotary members.

Accordingly, the engagement force of the main clutch is controlled at a position between the front and rear  
5 wheels of the four-wheel drive vehicle, by member of the electromagnetic force due to energization control corresponding to the accelerator or throttle position. The driving force output from the engine by way of the transmission can be transmitted to the wheels while a  
10 limitation is imposed on the differential motion of the differential interposed between the front and rear wheels, in accordance with or by member of the engagement force. Occurrence of unusual noise can be inhibited during such an operation, or vibration or physical shock, such as  
15 swinging of the vehicle body, can be dampened.

A seventh aspect of the present invention provides a power transfer apparatus of a four-wheel drive vehicle which is disposed between front and rear wheels and is capable of transmitting, to the wheels, driving force  
20 output from an engine by way of a transmission according to the engagement force by member of controlling engagement force of clutch member by the electromagnetic force derived from energization control, the power transfer apparatus comprising: gear shift status  
25 detection member for detecting the state of gear shift of

the transmission; and control member for subjecting an  
electromagnet to energization control such that the  
torque to be transmitted by the clutch member is set to a  
target value in accordance with a gear shift status  
5 signal detected by the gear shift status detection member.  
Hence, the driving force to be transmitted from the  
engine to the vehicle can be controlled optimally by  
controlling the clutch member in accordance with the gear  
shift state of the transmission. For instance, the  
10 advantages yielded by the first to sixth aspect of the  
present invention can also be yielded. For instance, the  
electromagnet can be subjected to energization control  
such that the transmission torque shifts to the target  
value, in accordance with a signal pertaining to the  
15 statuses of gear positions which are employed as  
substitutes for the gear positions of a manual  
transmission or those of an automatic transmission. As a  
result, the transmission torque can be controlled  
immediately under various situations; for example, where  
20 high drive torque is required while a vehicle remains at  
a low-speed state; where transmission torque higher than  
required is not needed for the vehicle to cruise at high  
speed; and where supply of transmission torque to four  
wheels is not required when the gear is shifted to a low-  
25 speed range (to prevent occurrence of interference with



an ABS), by member of increasing or decreasing the transmission torque required to achieve driving stability according to the gear position when the vehicle is taking corners.

5           Moreover, the above-mentioned object can also be achieved by a power transfer apparatus of a four-wheel drive vehicle, according to the another aspect of the present invention that comprises:

          a clutch that distributes a torque, that is  
10   generated by an engine via a transmission, between a primary wheel to a secondary wheel;

          means for detecting a gear shifting condition of the transmission; and

          a controller that receives an input signal  
15   indicating the gear shifting condition and generates a control signal that reduces a torque transmitted by the clutch to a target value.

          In addition, the above-mentioned object can also be attained by a device for controlling torque transmitted  
20   by a clutch, according to the further another aspect of the present invention that comprises:

          a controller that generates a control signal based on an input received from a transmission, wherein the control signal is output to the clutch to decrease the  
25   torque to a target value.

## BRIEF DESCRIPTION OF THE DRAWINGS

Fig. 1 is a skeleton plan view of a four-wheel drive vehicle according to an embodiment of the invention;

5        Fig. 2 is a cross-sectional view of a control coupling according to the embodiment;

Fig. 3 relates to the embodiment, wherein Fig. 3A is a graph showing a change in an accelerator or throttle position; Fig. 3B is a graph showing a change in the A/T  
10        range; and Fig. 3C is a graph showing a change in target torque;

Fig. 4 relates to the embodiment, wherein Fig. 4A is a graph showing a change in an accelerator or throttle position; Fig. 4B is a graph showing a change in the A/T  
15        range; and Fig. 4C is a graph showing a change in target torque;

Fig. 5 relates to the embodiment, wherein Fig. 5A is a table showing symbols used for ascertaining an A/T range; Fig. 5B is a table showing perception of direction  
20        of the A/T range; and Fig. 5C is a table for showing perception of inversion of the A/T range;

Fig. 6 relates to the embodiment, showing a table of operation settings employed when a determination is made as to whether or not the A/T range is inverted;

25        Fig. 7 relates to the embodiment, wherein Fig. 7A is

a table showing limitations on reduction of target torque, and Fig. 7B is a graph showing an operation image of limitation on reduction of the target torque;

Fig. 8 is a flowchart relating to the embodiment;

5 Fig. 9 shows an embodiment to which a manual transmission is applied, wherein Fig. 9A is a table showing signals employed for perceiving an M/T range, Fig. 9B is a table pertaining to perception of the direction of the M/T range, and Fig. 9C is a table showing  
10 perception of inversion of the M/T range;

Fig. 10 is a skeleton diagram showing an embodiment in which a control coupling is provided in a transfer section of an FR-based four-wheel drive vehicle;

Fig. 11 is a skeleton diagram showing an embodiment  
15 in which a control coupling is provided as a differential limiter for a center differential of a planetary differential gear mechanism of the FR-based four-wheel drive vehicle;

Fig. 12 is a skeleton diagram of a related-art four-  
20 wheel drive vehicle;

Fig. 13 is a semi-cross-sectional view of a related-art control coupling;

Fig. 14 describes drawbacks of the related art, wherein Fig. 14A is a graph showing a change in  
25 accelerator or throttle position, Fig. 14B is a graph

showing a change in the A/T range, and Fig. 14C is a graph showing a change in target torque; and

Fig. 15 describes drawbacks of the related art, wherein Fig. 15A is a graph showing a change in  
5 accelerator or throttle position, Fig. 15B is a graph showing a change in the A/T range, and Fig. 15C is a graph showing a change in target torque.

#### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

10 Figs. 1 and 2 relate to an embodiment of the present invention. Fig. 1 is a skeleton plan view of a four-wheel drive vehicle, and Fig. 2 is a cross-sectional view of a power transfer apparatus. The four-wheel drive vehicle shown in Fig. 1 is based on a front-engine and  
15 front-wheel (FF) configuration. A control coupling 1 serving as a power transfer apparatus is provided in a drive system disposed between front wheels 3 and rear wheels 5. In the embodiment, the control coupling 1 is interposed between a drive shaft 7 and a rear  
20 differential 9.

Driving force is transmitted from an engine 11 to the drive shaft 7 by way of an automatic transmission 13 serving as a transmission, and a transfer 15. The driving force is further transmitted to the rear wheels 5  
25 from the drive shaft 7 by way of the control coupling 1,

the rear differential 9, and an axle shaft 17. The driving force is further transmitted from the engine 11 to the front wheels 3 by way of the automatic transmission 13, the transfer 15, a front differential 19, and an axle shaft 21.

The control coupling 1 is controlled by a controller 23 serving as control means, by means of electromagnetic force derived from energization control corresponding to the accelerator or throttle position, thereby transmitting the driving force. The accelerator or throttle position is detected by means of an accelerator pedal position sensor 25. A detection signal output from the accelerator pedal position sensor 25 is delivered to the controller 23.

The automatic transmission 13 is provided with an automatic transmission controller 27. The detection signal of the automatic transmission controller 27 is used for indicating the position of a transmission shift lever determined through gear-shifting operation and in the embodiment is also delivered to the controller 23.

Accordingly, the controller 23 subjects the control coupling 1 to energization control in accordance with an accelerator or throttle position, by means of the detection signal output from the accelerator switch 25. The controller 23 also determines inversion and non-

inversion of the gear-shifting direction of the automatic transmission 13 in accordance with a signal output from the automatic transmission controller 27 (e.g., a range signal which will be described later), thereby effecting energization control of the control coupling 1, as will be described later. The controller 23 determines occurrence of inversion when switching has arisen from a drive range signal D to a reverse range signal R or from the reverse range signal R to the drive range signal D. If such switching does not arise, the controller 23 determines that the gear-shifting remains non-inverted.

The control coupling 1 assumes a specific configuration such as that shown in Fig. 2. As shown in Fig. 2, the control coupling 1 has an inner rotary member 29 and an outer rotary member 31. The inner and outer rotary members 29, 31 are coaxially provided so to be able to rotate in relation to each other. Interposed between the inner and outer rotary members 29, 31 are a main clutch 33 serving as clutch means, a pilot clutch 35, and conversion means 37 constituted of a cam mechanism. Torque is transmitted between the inner and outer rotary members 29, 31 in accordance with energization control.

The inner rotary member 29 is constituted of an axial member and is coupled synchronously to a drive pinion shaft 39 (Fig. 1) serving as an input shaft of the

rear differential 9. The outer rotary member 31 is constituted of a rotary case, and a coupling shaft 41 connected integrally on an end section of the outer rotary member is synchronously joined to a constant-  
5 velocity joint 43 (Fig. 1) disposed on the drive shaft 7.

The main clutch 33 is constituted of a plurality of friction plates which engage the inner rotary member 29 and the outer rotary member 31 and are alternately disposed in the axial direction. The pilot clutch 35 is  
10 constituted of a plurality of friction plates which engage a cam plate 45 of the conversion means 37 and the outer rotary member 31 and are arranged alternately.

In addition to being formed from the cam plate 45, the conversion means 37 is constituted of a pressure  
15 plate 47 and steel balls 49 interposed between a cam surface of the cam plate 45 and a cam surface formed on the pressure plate 47.

The cam plate 45 is disposed so as to be rotatable in relation to an outer peripheral surface of the inner  
20 rotary member 29. One friction plate of the pilot clutch 35 is spline-engaged with an outer peripheral surface of the cam plate 45. An inner peripheral side of the pressure plate 47 is spline-engaged with the inner rotary member 29, and an outer peripheral side of the pressure  
25 plate 47 axially opposes the friction plate disposed at

the end of the main clutch 33.

The pilot clutch 35 is fastened by means of an electromagnet 51 that attracts an armature 53. The electromagnet 51 is mounted on the outer rotary member 31  
5 and supported by an end plate member 54 interposed between the inner and outer rotary members 29, 31. The electromagnet 51 is connected to the controller 23 by way of a lead wire 55.

Accordingly, when the electromagnet 51 is subjected  
10 to energization control performed by the controller 23, the armature 53 is attracted by means of electromagnetic force of the electromagnet 51, whereupon the pilot clutch 35 is engaged. When driving force is transmitted from the outer rotary member 31 at this time, relative  
15 rotation arises between the inner and outer rotary members 29, 31. As a result, the cam plate 45, which attempts to rotate in conjunction with the outer rotary member 31, and the pressure plate 47, which attempts to rotate in conjunction with the inner rotary member 29,  
20 rotate relatively to each other by way of the pilot clutch 35. As a result, the pressure plate 47 is moved toward the main clutch 33 by means of the cam action developing between the steel balls 49 and the cam surface. In association with movement of the pressure plate 47,  
25 the main clutch 33 is engaged, whereby torque can be



transmitted from the outer rotary member 31 to the inner rotary member 29 by way of the main clutch 33.

Transmission of torque performed by the main clutch 33 can be arbitrarily controlled by means of energization control of the electromagnet 51 performed by the controller 23. Specifically, transmission of torque performed by the main clutch 33 enables energization control corresponding to the accelerator or throttle position.

Fig. 3 corresponds to Fig. 14. Fig. 3A is a graph showing a change in an accelerator or throttle position; Fig. 3B is a graph showing a change in the A/T range; and Fig. 3C is a graph showing a change in target torque (i.e., the value of a control current). A control current serves as implementation means when transmission torque of the control coupling 1 has achieved a target.

Fig. 3 shows a situation in which the vehicle is taking corners under the stipulation that a tight corner braking phenomenon will arise and in which, for example, the gear-shifting position of the automatic transmission 13 is switched from the drive range D to the reverse range R by means of performing tip-out and tip-in operations during the course of turns being taken, thereby reversing the gear-shifting direction.

In Fig. 3, the accelerator or throttle position is

set to an arbitrary closed (OFF) position by means of performing a tip-out operation at a point in time (1) before switching is made to the A/T range at a point in time (3). After switching has been made to the A/T range, the accelerator or throttle position is set to an arbitrary open (ON) position by means of performing a tip-in operation at a point in time (2). The friction factor  $\mu$  existing between the road surface and the tires corresponds to, e.g., a dry state.

10        The target torque of the control coupling 1 is controlled in a manner shown in Fig. 3C in accordance with the accelerator or throttle position shown in Fig. 3A and a change in the A/T range shown in Fig. 3B.

15        In Fig. 3, the gear-shifting direction is inverted at a point in time (3) during the course of the accelerator or throttle position being switched again to an open (ON) position by means of the tip-in operation performed at a point in time (2) after having once been switched to the arbitrary closed (OFF) position from the arbitrary open (ON) position by means of the tip-out operation at the point in time (1). When the accelerator or throttle position is switched from the ON position to the OFF position at the point in time (1), the controller 23 (see Fig. 1) performs energization control operation 25 by means of a change represented by a line segment F3, in

accordance with a signal output from the accelerator pedal position sensor 25 (see Fig. 1) until an inversion signal arises at the point in time (3).

The line segment F3 decreases the transmission torque of the main clutch 33, and controls energization such that the proportion of decrease in transmission torque is made steep at the beginning and then made gentle. The line segment F3 takes the proportion of decrease in transmission torque as being equivalent to a time constant. Hence, the embodiment adopts a multistage broken line. Alternatively, a curve can be made similar to a time-constant waveform by making control operation more elaborate.

Energization control is performed by means of such a line segment F3. In comparison with the control operation whose entirety assumes a steep slope which is described by reference to Fig. 14 and is based on the line segment F2, vibration of the control coupling 1 is dampened, thereby enabling shortening of a time required to decrease torque. As a result, physical shock such as swaying of the vehicle body can be inhibited without fail. Moreover, by virtue of the line segment F3 being made equivalent to the time constant, an amplitude can be made smaller in terms of vibration response. Hence, physical shock, such as swaying motion of the vehicle body, can be

made smaller in a more reliable manner.

At the point in time (3) located between the point in time (1) for the tip-out operation and the point in time (2) for the tip-in operation, the controller 23  
5 determines occurrence of inversion of the gear-shifting direction from the signal output from the automatic transmission controller 27 (see Fig. 1). Energization control of the electromagnet 51 is performed such that transmission torque of the main clutch 33 is decreased to  
10 a target value in accordance with the inversion signal.

The target value is intended for use in bringing target torque to, e.g., 0 Nm. Energization control is performed in accordance with a line segment F4 having a steep slope until target torque 0 Nm is achieved. The  
15 energization control is maintained from the time when control of transmission torque to 0 Nm starts until the time when time "t" (sec.) lapses. Here, the constant time "t" (sec.) is set as a response time during which actual transmission torque of the control coupling 1  
20 becomes close to the target torque (0 Nm), by means of energization control of the electromagnet 51 in accordance with the line segment F4. Accordingly, even when energization control is performed such that the control coupling 1 assumes target torque by means of the  
25 tip-in operation performed at the point in time (2) after

the gear-shifting direction has been inverted at the point in time (3), the control coupling 1 can bring the transmission torque to 0 Nm without fail before the transmission torque attains a target torque of a lock  
5 mode.

Even when the direction of torsion of the drive system attributable to the tight corner braking phenomenon has been changed by means of inversion of the gear-shifting direction, the direction of generated  
10 driving force can be changed with the transmission torque of the control coupling 1 being held at 0 Nm, thereby enabling smooth transition of the transmission torque to the target torque. Accordingly, since the torque (force) produced at the time of elimination of backlash is made  
15 small by means of a change in the direction—in which backlash is eliminated—due to a change in the direction of torsion of the drive shaft along the axis of the control coupling 1, the backlash is eliminated comparatively gradually, thereby considerably reducing or  
20 avoiding occurrence of impact sound due to elimination of backlash. Therefore, unusual sound or physical shock, which would otherwise grate on the occupants' ears, can be inhibited without fail.

In this case, subjecting the area of the line  
25 segment F3 to energization control at a slope like a line

segment F4 results in occurrence of great physical shock, such as that induced conventionally. Hence, countermeasures must be taken before and after inversion of the gear-shifting direction arising at the point in time (3). In the embodiment, an appropriate one is selected from the line segments F3 and F4 according to a purpose, by means of detecting inversion of the gear-shifting direction arising at the point in time (3). Occurrence of physical shock and that of unusual sound can be inhibited simultaneously without fail.

Fig. 4 corresponds to Fig. 15, wherein Fig. 4A is a graph showing a change in an accelerator or throttle position; Fig. 4B is a graph showing a change in the A/T range; and Fig. 4C is a graph showing a change in target torque (i.e., the value of a control current).

Fig. 4 shows that the four-wheel drive vehicle comes to a halt while taking corners for reasons of a tight corner braking phenomenon and that the gear-shifting direction is inverted during the course of the tight corner braking phenomenon. Here, the friction factor  $\mu$  existing between the tires and the road surface is presumed to be high.

Depending on a vehicle, when the value of the accelerator or throttle position is below a predetermined value, the vehicle often comes to a halt because of the

tight corner braking phenomenon. Therefore, if in this state the A/T range is switched from the drive range D to the reverse range R at a point in time (4) as shown in Fig. 4B, to thereby reverse the gear-shifting direction, an inversion signal is detected at the point in time (4) in the same manner as mentioned previously. The controller 23 subjects the electromagnet 51 to energization control along the line segment F4 having a steep slope, in accordance with the inversion signal.

10 The energization control is maintained for a given period of time "t" (sec.) from the point in time (4). Subsequently, the electromagnet 51 is subjected to energization control such that the control coupling 1 assumes a target torque. As a result, the transmission

15 torque of the control coupling 1 decreases to a target value; e.g., 0 Nm.

The target control current value is maintained for a given period of time at "t" (sec.). As mentioned above, there can be ensured a time from the time when the electromagnet 51 is subjected to energization control until the time when the transmission torque of the control coupling 1 actually decreases to the target value. Accordingly, as in a case shown in Fig. 3 where the line segment F4 and the time "t" (sec.) are ensured, in the

25 embodiment shown in Fig. 4 there can be reliably

prevented occurrence of unusual sound or physical shock, which would otherwise grate on the occupants' ears for reasons of a change in the direction in which backlash is eliminated.

5           Inversion of the gear-shifting direction is not limited to a change from the drive range D to the reverse range R as mentioned above; the same also applies to a case where the gear-shifting direction is switched from the reverse range R to the drive range D.

10           Fig. 5 shows elements to be used for determining occurrence of inversion of the gear-shifting direction. Fig. 5A is a table showing signals used for ascertaining an A/T range; Fig. 5B is a table showing perception of direction of the A/T range; and Fig. 5C is a table for  
15   showing perception of inversion of the A/T range.

As shown in Fig. 5A, the parking range P, the reverse range R, the neutral range N, and the drive range D are used for perception of the A/T range, and the automatic transmission controller 27 detects the A/T  
20   range. As shown in Fig. 5B, binary number 0 is assigned to the reverse range R, and binary number 1 is assigned to the drive range D. Binary numbers 1, 0 are not fixedly assigned to a parking range P and a neutral range N; when the parking range P or the neutral range N comes  
25   into effect, the binary number assigned to the previous



range is retained as is. For instance, when switching is made from the reverse range R to the neutral range N, the binary number assumes 0. In contrast, when switching is made from the drive range D to the neutral range N, the  
5 binary number assumes 1.

By means of an exclusive OR result determined from the binary number 1 or 0 assigned to the previous range perceived by switching of the A/T range and from the binary number 1 or 0 assigned to the current range,  
10 inversion of the gear-shifting direction is perceived. Specifically, if the previous range assumes binary number 0 and the current range assumes binary number 0, or if the previous range assumes binary number 1 and the current range also assumes binary number 1, 0 is  
15 perceived as shown in Fig. 5C, and therefore no inversion of the gear-shifting direction is perceived. In contrast, when the previous range assumes binary number 1 and the current range assumes binary number 0, or if the previous range assumes binary number 0 and the current range  
20 assumes binary number 1, occurrence of inversion of the gear-shifting direction is perceived.

Fig. 6 is a table showing operation settings employed when a determination is made as to whether or not the A/T range is inverted. Fig. 6 shows setting of  
25 target torque on the basis of determination of inversion

of a range in a 4WD mode, a time during which the target torque is maintained, limitations on reduction of the target torque, and limitations on increase in the target torque. In this case, the target torque represents a  
5 constant related to a target control value; that is, a control current value, to be employed for bringing the control coupling 1 to the target torque.

As described by reference to Figs. 3 and 4, when inversion of the range is determined to have been  
10 performed, target torque T (to be converted to a constant) is set. Concurrently, the time "t" during which the target torque is to be maintained (to be converted to a constant) and a limitation L on reduction of the target torque (to be converted to a constant) are set. As a  
15 result, the line segments F3, F4 shown in Figs. 3 and 4 are determined.

Even when inversion of the range is determined not to have been performed, L1, L2, and L3 (all to be converted to constants) are set as limitations on  
20 reduction of the target torque. The line segment F3 shown in Fig. 3 is set.

Fig. 7 describes settings of the line segment F3 shown in Fig. 3. Fig. 7A is a table showing the constants L1, L2, and L3 to be used for limiting the  
25 reduction of the target torque at respective stages, and

Fig. 7B is a graph showing the line segment F3.

As shown in Figs. 7A and 7B, the period from the point in time (1) (see Fig. 3) for tip-out to the point in time (3) (see Fig. 3) where an inversion signal arises is divided into three stages; that is, a first stage in which the target torque exceeds  $T_2$ ; a second stage in which the target torque exceeds  $T_1$  and goes below  $T_2$ ; and a third stage in which the target torque goes below  $T_1$ . In each of the stages, the constants  $L_1$ ,  $L_2$ , and  $L_3$  are set. As a result, control represented by the line segment F3 as shown in Fig. 7B can be performed.

Fig. 8 is a flowchart showing control operations to be performed depending on whether or not the gear-shifting direction is inverted. When processing pertaining to the flowchart shown in Fig. 8 is performed, processing "reading of gear shift position signals produced before and after occurrence of a shift" is executed in step S1. Through the processing, binary numbers 0, 1 issued before and after occurrence of a shift in transmission in the respective ranges shown in Fig. 5 are read, whereupon processing proceeds to step S2.

In step S2, a determination is made as to whether or not inversion has been performed, by means of an exclusive OR result shown in Fig. 5C. When inversion is determined to have been performed, processing proceeds to

step S3. In contrast, when inversion is determined not to have been performed, processing proceeds to step S4.

In step S3, processing "selection of a method A for reducing control torque" is performed. Through the  
5 processing, the line segment F4 shown in Figs. 3 and 4 is selected by means of setting the constants; that is, the target torque T, the retention time "t," and the reduction limitation L. Then, processing proceeds to step S5.

In step S5, the electromagnet 51 (see Fig. 2) of the  
10 control coupling 1 (see Fig. 1) is subjected to energization control along the line segment F4 set by the constants T, "t," and L, and processing proceeds to step S6.

In step S6, processing "timer start" is performed,  
15 whereby the timer is set by the controller 23 (see Fig. 1). Processing then proceeds to step S7. The time set in step S6 is a constant time "t" (sec.).

In step S7, processing "timer count-up" is performed, whereby a timer performs count-up operation in increments  
20 of preset time. Then, processing proceeds to step S8.

In step S8, a determination is made as to whether or not a time has elapsed. This determination is intended to be made as to whether or not the constant time "t" (sec.) has elapsed. If the time has not yet elapsed  
25 (when NO is selected), processing returns to step S7. If

the time has elapsed (when YES is selected), processing proceeds to step S9. Therefore, if the constant time "t" (sec.) has not elapsed, a count-up operation is performed in step S7 during the course of lapse of the constant  
5 time. Energization control of the electromagnet 51 along the line segment F4 is continuously performed. If the constant time "t" (sec.) has elapsed, processing "timer clear" is performed in step S9. The timer is reset, and processing is again iterated from step S1.

10 In step S4, processing "selection of a method B for reducing control torque" is performed. Through the processing, the line segment F3 shown in Fig. 3 is selected by means of setting of the constants T1, T2, L1, L2, and L3. Processing then proceeds to step S10.

15 In step S10, processing "coupling control" is performed, and the electromagnet 51 of the control coupling 1 is subjected to energization control according to the line segment F3. As a result, a reduction in control torque is performed in preference to operability,  
20 thereby inhibiting occurrence of physical shock and unusual noise.

The embodiment has described a case where the transmission is an automatic transmission. The same also applies to a manual transmission.

25 Fig. 9 corresponds to Fig. 5 and relates to a method

for determining whether or not the gear-shifting direction is inverted by the manual transmission. Fig. 9A is a table showing signals employed for perceiving a shift position of the manual transmission (i.e., an M/T range); Fig. 9B is a table showing perception of a direction of the M/T range; and Fig. 9C is a table showing perception of inversion of the M/T range.

As shown in Fig. 9A, detection signals pertaining to the neutral range N, the reverse range R, the first range, the second range, and the third range are used for perception of the M/T range. Detection of ranges is performed by means of shift switches or the like provided for respective ranges around the shift lever.

As shown in Fig. 9B, numeral 1 is fixedly assigned to the reverse range R; and numeral 0 is fixedly assigned to forward directions 1<sup>st</sup> to X such as the first range, the second range, the third range, etc. No fixed value is assigned to the neutral range N, and a value identical with that employed for the previous range is maintained. If the previous range is the reverse range R, the neutral range N assumes a value of 1. If the previous range is any one of 1<sup>st</sup> to X, such as the first range, the second range, the third range, etc., the neutral range assumes a value of 0. As a result, as in the case of Fig. 5, when the exclusive OR result assumes a value of 0, no

inversion can be determined to have been performed in the manner as shown in Fig. 9C. When the exclusive OR result assumes a value of 1, inversion can be determined to have been performed.

5           Accordingly, the electromagnet 51 is subjected to energization control in the same manner through use of perception of inversion. Occurrence of physical shock or unusual noise can be inhibited accurately.

          The front-engine, front-drive (FF) four-wheel drive  
10 vehicle has been described in the embodiment. However, the invention can also be applied in the same manner to a front-engine, rear-drive (FR) four-wheel drive vehicle such as that shown in Figs. 10 and 11. Constituent elements corresponding to those shown in Fig. 1 are  
15 assigned corresponding reference numerals and will be described.

          In Fig. 10, a control coupling 1A is provided within a transfer 15A disposed at an output-side of the automatic transmission 13. An inner rotary member 29A of  
20 the control coupling 1A is coupled to the output-side of the automatic transmission 13 and further joined to the drive shaft 7. An outer rotary member 31A of the control coupling 1A is provided with a gear 59. An interlocking belt 65 is passed between the gear 59 and a gear 63  
25 provided on an output shaft 61 of the transfer 15A. The

output shaft 61 is coupled to an input shaft 69 of a front differential 19A by way of a shaft 67.

Accordingly, driving force is transmitted from the engine 11 to the rear wheels 5 by way of the automatic transmission 13, the inner rotary member 29A of the control coupling 1A, the drive shaft 7, the rear differential 9, and the axle shaft 17. In the meantime, the driving force can also be transmitted to the front wheels 3 from the inner rotary member 29A of the control coupling 1A by way of the control coupling 1A, the gear 59, the interlocking belt 65, the gear 63, the output shaft 61, the shaft 67, the input shaft 69, the front differential 19A, and the axle shaft 21.

Accordingly, a 4WD mode is controlled by means of energization control analogous to that to which the control coupling 1A is to be subjected, thereby accurately avoiding occurrence of physical shock and unusual noise.

Fig. 11 shows a front-engine, rear-drive (FR) four-wheel drive vehicle to which the invention is applied. The control coupling 1, which performs direct transmission of driving force, is not subjected to energization control in the manner mentioned above. The main clutch 33, the pilot clutch 35, the electromagnet 51, or the like are provided so as to limit differential



operation of a center differential 71 by means of a planetary differential gear mechanism of a transfer 15B. Specifically, a planetary gear 75 of the center differential 71 is supported by a shaft 73 coupled to the output-side of the automatic transmission 13. An inner rotary member 29B of a control coupling 1B is coupled to a sun gear 77 of the center differential 71.

A gear 59B is coupled to the inner rotary member 29B, and an interlocking belt 65B is passed between the gear 59B and a gear 63B of an output shaft 61B of the transfer 15B. The output shaft 61B is coupled to an input-side of the front differential 19 by way of a shaft 67B.

An outer rotary member 31B of the control coupling 1B is provided integrally on a link gear 79 of the center differential 71 and further coupled to the drive shaft 7 by way of an output shaft 81 of the transfer 15B.

In the embodiment shown in Fig. 11, the driving force output from the engine 11 by way of the automatic transmission 13 is transmitted to the front differential 19 from the sun gear 77 by way of the inner rotary member 29B, the gear 59B, the interlocking belt 65B, the gear 63B, the output shaft 61B, and the shaft 67B, as a result of the sun gear 77 and the ring gear 79 being rotated together by means of revolution of the planetary gear 75 of the center differential 71 from the shaft 73. The

driving force is also transmitted to the rear differential 9 from the ring gear 79 of the center differential 71 by way of the outer rotary member 31B, the output shaft 81, and the drive shaft 7.

5           If a difference arises between the rotational speed of the front wheels 3 and that of the rear wheels 5 because of turning of corners during the course of transmission of driving force, the planetary gear 75 rotates in accordance with a difference in rotations of  
10 the inner rotary member 29B and those of the outer rotary member 31B, and the center differential 71 absorbs the difference in rotation.

          In accordance with high-speed travel, low-speed travel, cornering, travel over a poor road, etc., the  
15 electromagnet 51 is subjected to energization control, thereby controlling the engagement force of the pilot clutch 35 and controlling the transmission torque of the control coupling 1.

          Through control of the transmission torque of the  
20 control coupling 1, relative rotation existing between the inner rotary member 29B and the outer rotary member 31B is controlled from a totally free state to a locked state, thereby enabling control of differential rotation of the center differential 71.

25           Even in such a structure, the physical shock

developing in the control coupling 1 is dampened by means of similar control operation, thereby inhibiting occurrence of unusual noise.

5       The foregoing embodiment shows that a processing signal and a range signal produced by an automatic transmission controller are used as inversion detection means for detecting inversion and non-inversion of the gear-shifting direction of the transmission. A signal output directly from an inhibitor switch provided in the  
10       automatic transmission controller can also be used as another detection means. Moreover, a signal output directly from the inhibitor switch can also be used as detection means separate from the automatic transmission controller.

15       Here, the inhibitor switch is specifically illustrated. A contact signal of the inhibitor switch is utilized as detection means. In accordance with a combination of connection and non-connection states of a plurality of contact signals, a determination can be made  
20       as to whether inversion or non-inversion has arisen.

      Further, the automatic transmission can use a signal pertaining to the position of a shift lever or a rotation angle signal of a gear-shifting rotary drum as detection means, regardless of a signal pertaining to the position  
25       of a manual valve, the automatic transmission, and the

manual transmission.

The invention is not limited to the form of the transmission or specific detection means. Various transmissions or various detection means (i.e., a  
5 detection signal etc.) are selected as required within the scope of the invention and can be adopted in combination.

While there has been described in connection with the preferred embodiment of the invention, it will be obvious  
10 to those skilled in the art that various changes and modifications may be made therein without departing from the invention, and it is aimed, therefore, to cover in the appended claim all such changes and modifications as fall within the true spirit and scope of the invention.

15 In the embodiment, control is performed in accordance with detection of inversion of the transmission. However, clutch means can be controlled in accordance with the state of gear shift of the transmission, to thereby appropriately control the  
20 driving force to be transmitted from the engine to the vehicle, by means of providing a vehicle with a gear shift status detection means for detecting the state of gear shift of the transmission and control means for subjecting the electromagnet to energization control such  
25 that the transmission torque of the clutch means is

controlled to a target value in accordance with a gear shift status signal detected by the gear shift status detection means. Such a configuration, among others, can also yield the same effect as that yielded by the

5 embodiment. Moreover, an electromagnet can be subjected to energization control such that transmission torque shifts to a target value, in accordance with signals pertaining to respective states of gear shift employed as substitutes for a gear position of the manual

10 transmission or a gear position of the automatic transmission. As a result, the transmission torque can be controlled immediately under various situations; for example, where high drive torque is required while a vehicle remains in a low-speed state; where transmission

15 torque higher than required is not needed for the vehicle to cruise at high speed; and where supply of transmission torque to four wheels is not required when the gear is shifted to a low-speed range (to prevent occurrence of interference with an ABS), by means of increasing or

20 decreasing the transmission torque required to achieve driving stability according to the gear position when the vehicle is taking corners.